

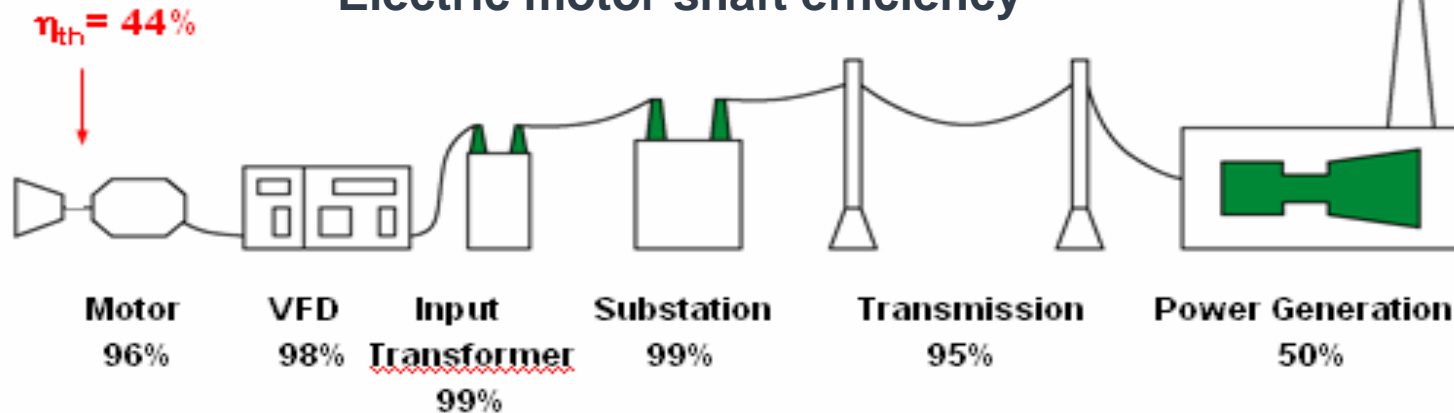
# **Why Electric Compression or otherwise why still Gas Engine driven Compression**

**Gas/Electric Partnership  
Special Workshop  
Electric Compression Economics,  
August 27 2009, Houston**

# Overall efficiency comparison

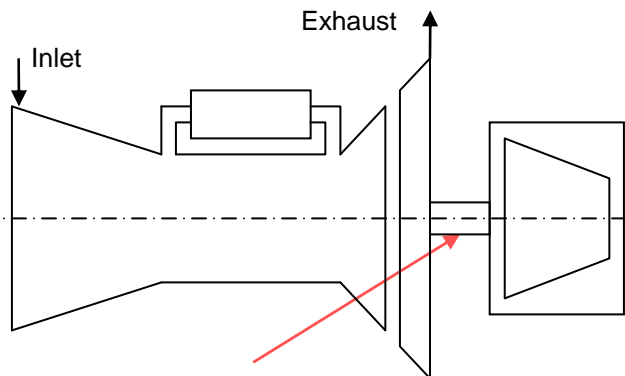


## Electric motor shaft efficiency



**Thermal Efficiency**  $\eta_{th} = (0.96)(0.98)(0.99)(0.99)(0.95)(0.50) = 44\%$

## Gas turbine shaft efficiency



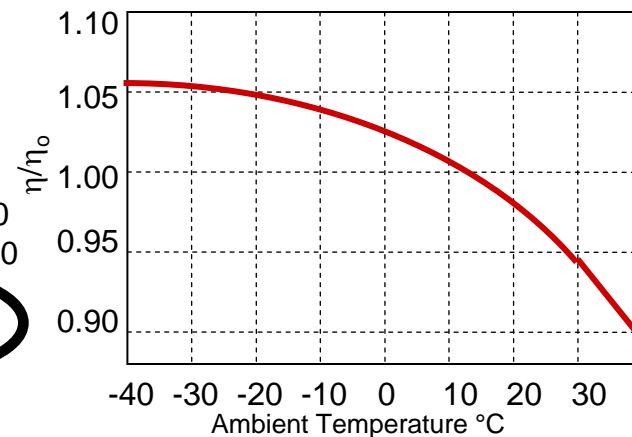
ISO Efficiency  
15°C, 1 Bar (100 kPa)

Best available ISO efficiency ~ 36%

Corrections for

- Inlet & exhaust systems: 0.98
- Auxiliary loads: 0.99
- Temperature (-40 to +40°C) 1.05 - 0.90
- Part load: 1.00 - 0.60

**Actual shaft efficiency** 36% - 19.0%



# Fundamental changes in the energy market



## The situation until late '80s

Since the gas turbine demonstrated its reliability in industrial applications in the late '50s/early '60s, it has been the most widely used driver for gas pipeline applications in the mid to large power ranges. Smaller power requirements have been shared with gas engine reciprocating compressors. Although electric drives have long been commonplace on liquid lines, until recently they have rarely, if ever, been used for gas pumping. The reasons for this are well known:

- The gas pipeline/storage companies' reluctance to be dependent upon an external source of power being supplied by a competing industry
- The strongly regulated electricity and gas market
- The cost of electric power compared with gas.

**The situation since early '90s** has undergone fundamental changes in a process which is still ongoing in North America and in Europe.

- Deregulation of the gas and electric utilities has on the one hand caused the pipeline companies to make their systems more efficient and on the other has led to a significant reduction in the cost of electric power.
- The move from nuclear to gas fired power stations has made the two industries become more dependent upon each other. (present situation in europe)
- Gas and electricity are converging commodities and futures in both are traded on stock exchanges. This development has led to the formation of energy corporations as opposed to gas or electricity utilities. (present situation in europe)

# Technologies Gas Transport / Storage market



## Gas Transport Application

- Gas Turbine driven Centrifugals
- Gas Engine RECIP's
- Electric Motor driven Centrifugals
- Electric Motor driven RECIP's

## Gas Storage Application

- Gas Engine RECIP's
- Electric Motor driven RECIP's
- Electric Motor driven Centrifugals



## Gas Transport Application

- Electric Motor driven Centrifugals
- Gas Turbine driven Centrifugals
- Electric Motor driven RECIP's

## Gas Storage Application

- Electric Motor driven Centrifugals
- Gas Turbine driven Centrifugals
- Electric Motor driven RECIP's

**clear tendency to EM driven centrifugals,  
RECIP's only used in case flow's are  
too small for centrifugals**

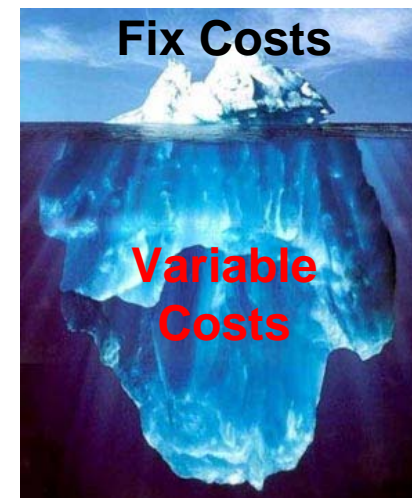
# Fundamental evaluation criteria's to make the decision for Gas Engine/Electric Compression



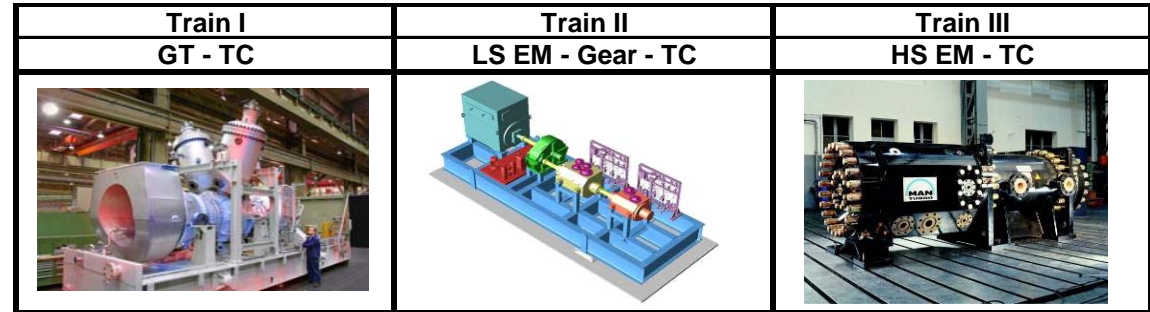
**Economical evaluation based on a full Life Cycle Cost analysis has to be performed for all alternative technologies**

- **CAPEX- Analysis:** considering all overall installation costs, like costs for compression train, building, grid connection Install.&Comm. ,Permit
- **OPEX-Analysis:** costs for energy, personnel, buildings, emmissions, interest rates, downtime
- **MAEX-Analysis:** spare parts, personnel, downtime

The following spreadsheet shall be regarded as a guideline, which aspects have to be considered for such an overall **qualitive** LCC analysis from equipment suppliers point of view.



# CAPEX Comparison of Different Turbo Compressor Train Concepts



## CAPEX for Compressor train

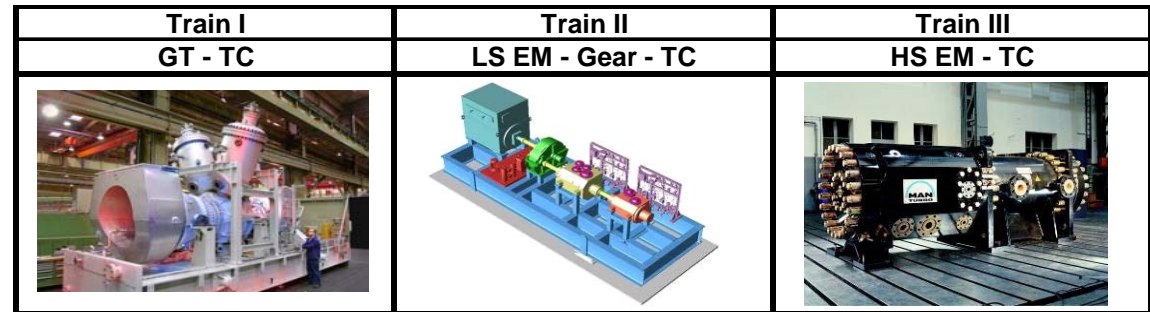
Centrifugal Compressor	0	0	0
Driver (GT, VSDS, EM)	-2	1	-1
Gear / (Gear not needed)	-1 (2)	-1	(2)
Bearing System	0	0	-2
Lube/Control/Working Oil system	-1	-1	2
Dry Gas Seal System	-1	-1	2
(Capital) Spare Parts	-1	-1	1

## CAPEX for Plant

TC Building (area / height) hazardous area	-2	1	2
VFD Building (area / height) non hazardous	2	-1	-1
Power Supply Installation (Fuel Gas Piping / Electrical Cable)	1	-1	-1
Oil System Air Cooler	-1	-1	2
Equipment Installation & Commissioning	-2	1	2

weighting of train characteristics with each other in respect to capital investment costs (+2 most beneficial, -2 less beneficial)

# CAPEX Comparison of Different Turbo Compressor Train Concepts ... continued



## CAPEX for Building

Foundation Size / Civil Works	-1	1	2
Interconnecting Piping of Lube / Control / Working Oil to Air Cooler	-1	1	2
Noise mitigation	-2	-1	1
Interconnecting Piping DGS	-1	-1	2
Crane Capacity	-1	1	2

## CAPEX for Auxiliaries

Intake Filter System	-2	2	2
Fire and Gas Detection System	-1	1	1
Emission Control	-1	1	2
Harmonic Filters (LCI only)	2	(-2) 2	2


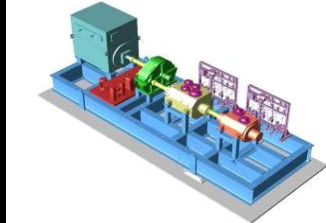

## Environmental related costs

VFD Cooling System Noise Emission	2	-1	-1
Oil Cooling System Noise Emission	-1	-1	2
Fulfillment of emission regulations	-2	1	2
Emission Credits	-2	1	1
Permitting Costs	-1	1	2

weighting of train characteristics with each other in respect to capital investment costs (+2 most beneficial, -2 less beneficial)

# OPEX/MAEX Comparison of Different Turbo Compressor Train Concepts


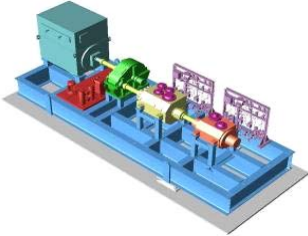



Train I GT - TC	Train II LS EM - Gear - TC	Train III HS EM - TC
		

Train full load cycle efficiency		
LOW due to low driver efficiency	HIGH due to high driver efficiency	HIGH due to high driver efficiency
Train part load efficiency		
LOW due to low driver efficiency. All trains experience reduced efficiency in throttling mode (recycle) at very low flow.	HIGH driver efficiency is nearly constant at varying power levels. All trains experience reduced efficiency in throttling mode (recycle) at very low flow.	HIGH driver efficiency is nearly constant at varying power levels. All trains experience reduced efficiency in throttling mode (recycle) at very low flow. Less critical due to increased speed range
Restart capability after forced shutdown and start density		
2 to 3 hour delay for restart due to driver thermal soak. Limited number of restarts per hour.	Restart ability varies with motor design. Limited number of restarts per hour.	Immediate restart. No limit on number of starts per hour.
Load assumption		
Approximately one-half hour after successful start.	Ability to start and come to full speed under load.	Ability to start and come to full speed under load.
Sensitivity to site conditions		
Available power reduced with increase in elevation, temperature, and humidity	Full power available at all site conditions	Full power available at all site conditions

# OPEX/MAEX Comparison of Different Turbo Compressor Train Concepts ... continued


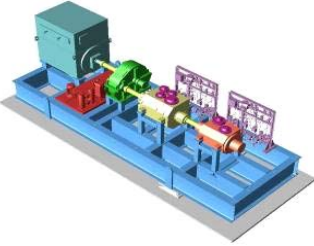



Train I GT - TC	Train II LS EM - Gear - TC	Train III HS EM - TC
		

<b>Controllability</b>		
Reasonably good, but depends on firing temperature and heat distribution	Speed controllability less accurate due to frequency adjustment range 0-60Hz(+/- 0.5 Hz) (200 rpm)	Excellent speed controllability due to high frequency operation at about 200 Hz (+/- 0.5Hz) (30 rpm)
<b>Ease of remote operation</b>		
Good	Better	Excellent ( all electric concept)
<b>Speed control range</b>		
typically 70 - 105%	typically 70 - 105%	30 - 105%
<b>Sarting reliability</b>		
Good	Excellent	Excellent
<b>Auxiliary consumption</b>		
Lube oil, lube oil filter media, inlet air filter media, power for lube oil pumps, lube oil cooler fans. Nitrogen required for seal gas buffering. Instrument air consumption, for filter/purge pulse cleaning.	Lube oil filter media, power for lube oil pumps, lube oil cooler fans. Nitrogen required for seal gas buffering. Nitrogen required for seal gas buffering. Instrument air consumption, for motor purge and motor pressurisation needed	Cooling gas filter media. Reduced instrument air consumption due usage of only one TCV for motor cooling. No air purge requirements. AMB power supply.

# OPEX/MAEX Comparison of Different Turbo Compressor Train Concepts... continued



Train I GT - TC	Train II LS EM - Gear - TC	Train III HS EM - TC
		

<b>Eshhaust emissions</b>		
Seal gas must be vented or flared; CO2 and NOx emission.	Seal gas must be vented or flared.	Zero local emissions.
<b>Maintenance costs</b>		
higher	lower	lowest
<b>Unit availability</b>		
lowest	high	highest
<b>Unit reliability</b>		
lowest	high	highest
<b>Ability to trade carbon credits</b>		
lowest	high	highest
<b>Fuel/Energy costs</b>		
Function of local cost of electricity		
<b>Risk of stricter Furure Emission Regulations</b>		
high	low	low
<b>Insurance risk</b>		
higher	lower	lower

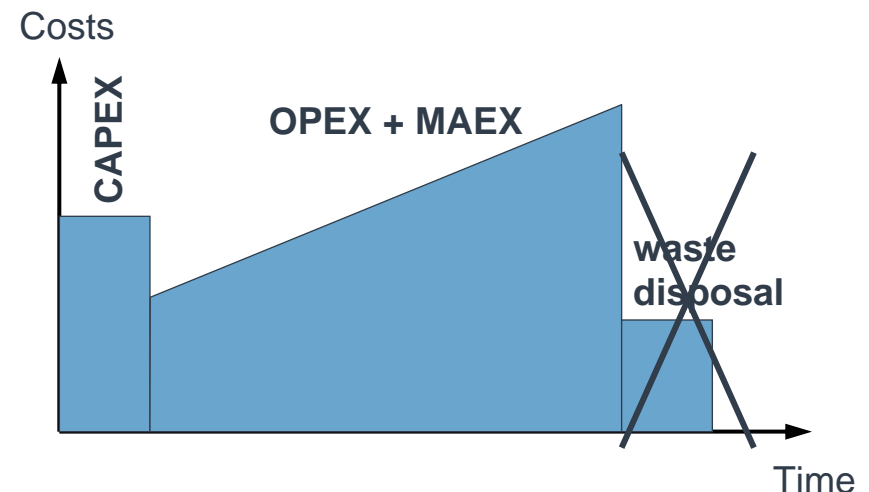
# Life Cycle Costing: Definition



## ■ Common Definition

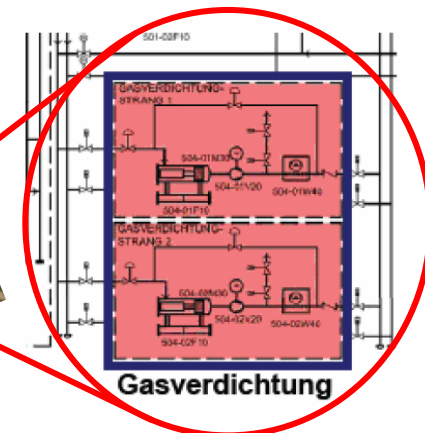
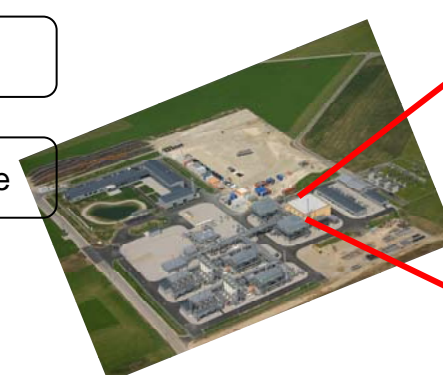
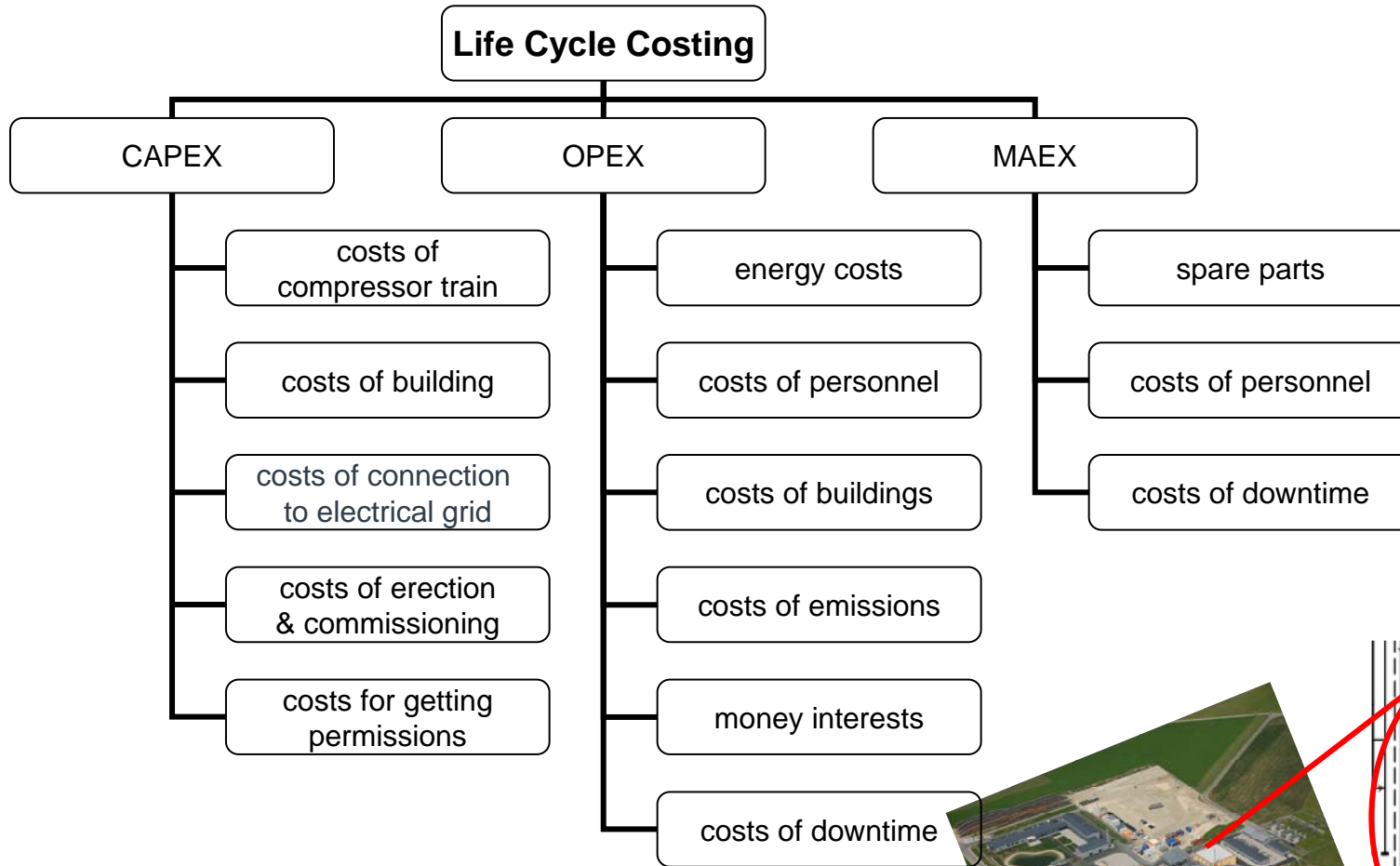
Classification and summation of single cost factors of a examined object within its time of life into following fields

- acquisition (CAPEX)
- operation and maintenance (OPEX + MAEX)
- waste disposal (not considered)



- used definition for the present analysis :  $LCC = CAPEX + OPEX + MAEX$

# Life Cycle Costing Analysis

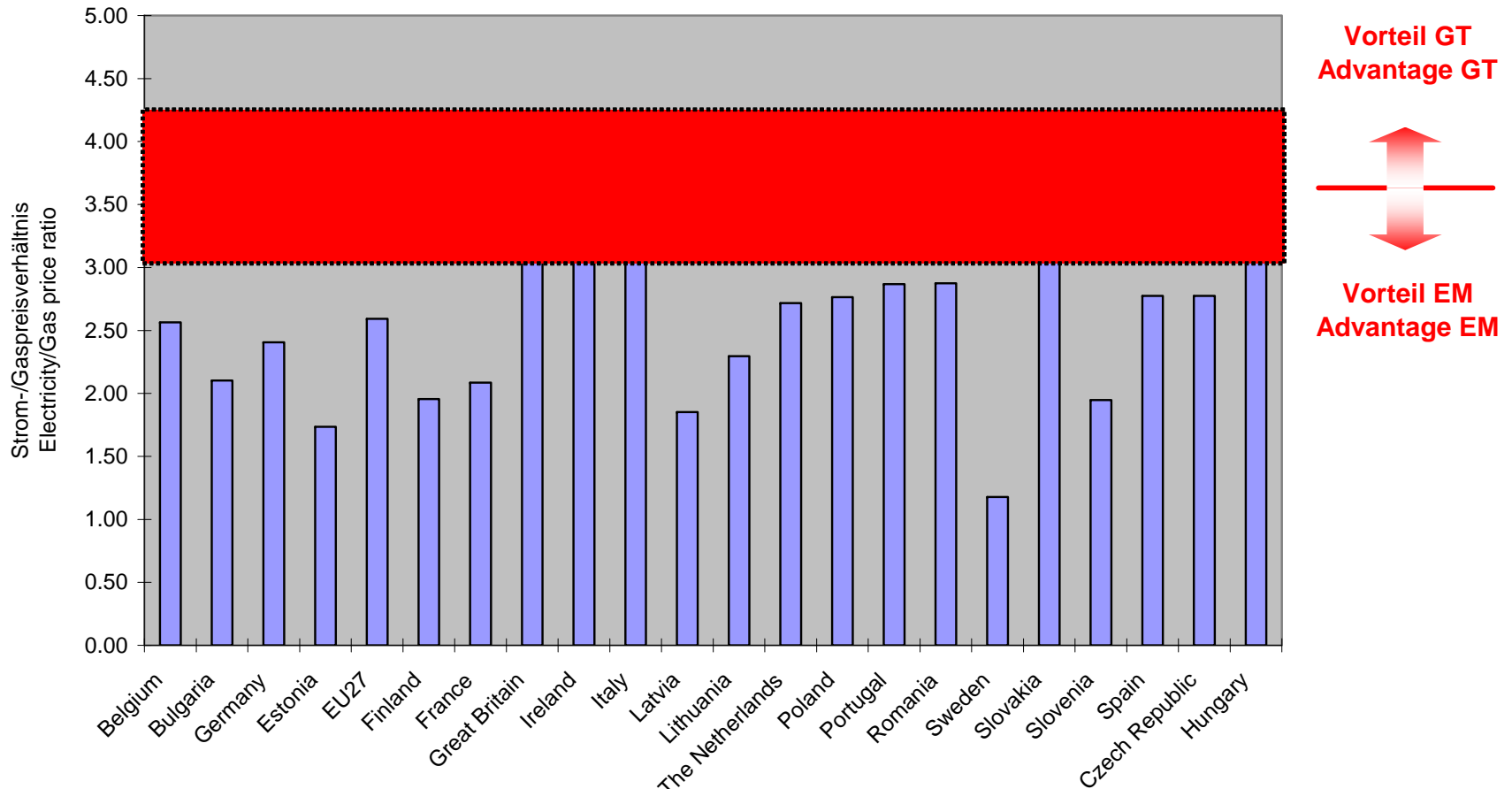


# Energy Prices



## ■ Electricity/Gas price ratio Europe

(The ratio decreased during 2008 due to heavy price fluctuations for oil and gas.)



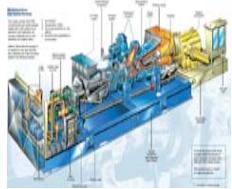


Prices from Eurostat, Industry, 2008s1, incl. excise tax, excl. VAT

# Case Study: Input data



## Life Cycle Costing - Analysis Gas storage



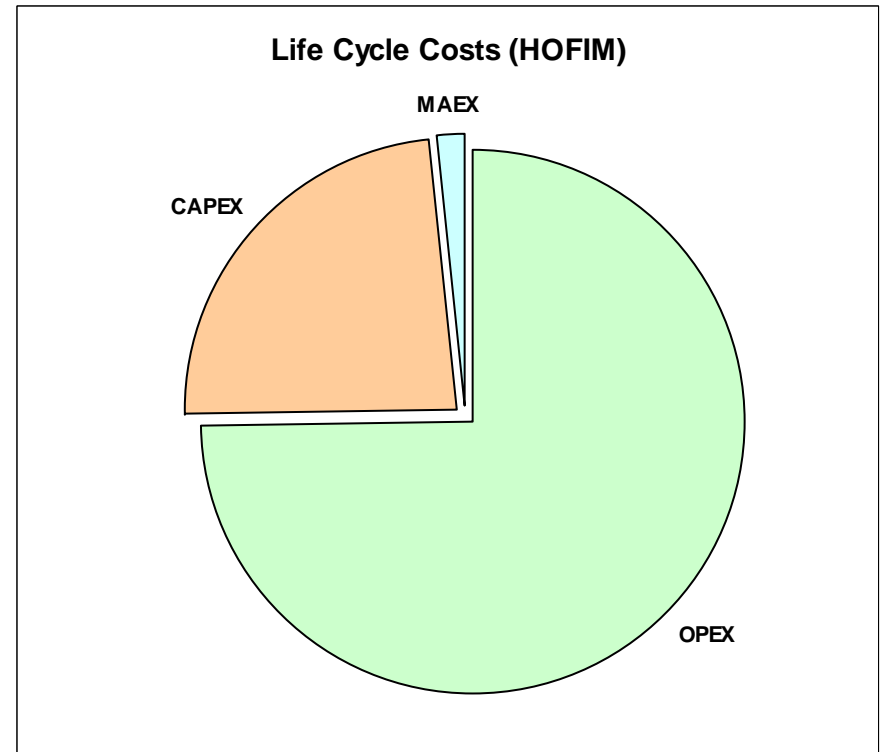
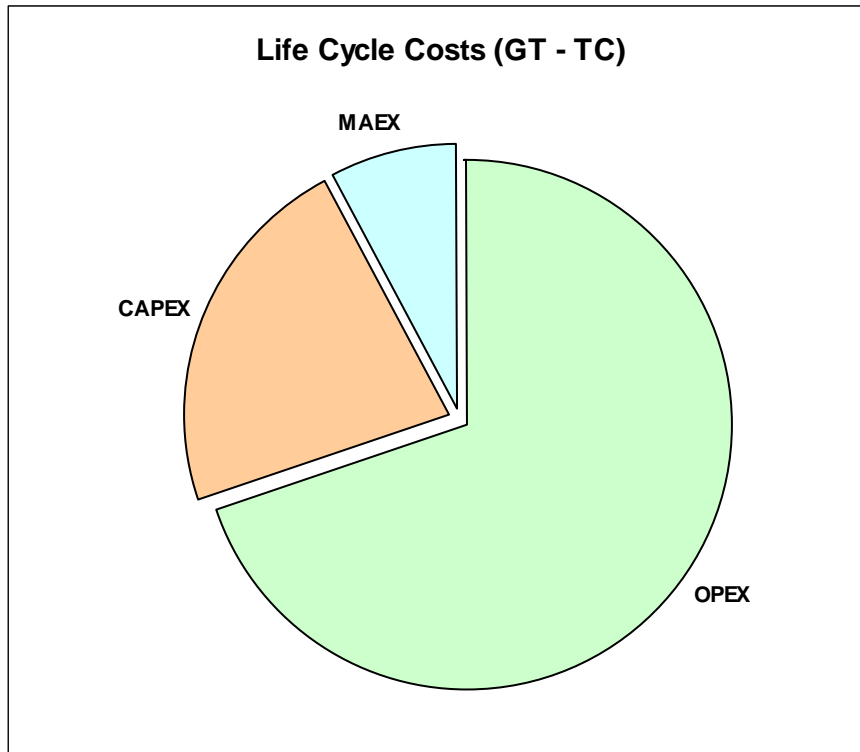
Compressor String Concept		GT - Gear - TC	LS EM - Gear - TC	HOFIM
Distance to network plane	6 [km]			
Redundancy Concept (N+R)	1+1			
Country	Great Britain			
Storage Type *)	Depleted Gas Field			
AGV *)	0 [Nm <sup>3</sup> ]			
Injection rate	80'000 [Nm <sup>3</sup> /h]			
Withdrawal rate	80'000 [Nm <sup>3</sup> /h]			
<div style="display: flex; justify-content: space-around;"> <div style="border: 1px solid black; padding: 5px;">Show total efficiency</div> <div style="border: 1px solid black; padding: 5px;">Calculate LCCs</div> </div>				
<b>1</b>				
<b>2</b>				
Program version 3.2 dated 2009-06-10				
Yearly start/stop frequency of the compressor facility	[-]	50		
Electricity price (industry, 2008S01, excl. VAT)	[€/kWh]		0.08530	
Gas price (industry, 2008S01, excl. VAT)	[€/kWh]	0.026		
Installed Power	[kW]	6'000	6'000	6'000
Technical availability of a string	[%]	95.0	98.0	98.5
Availability according to redundancy concept	[%]	99.750	99.960	99.978
Total efficiency of the compressor station	[%]	27.9	72.5	71.2
Yearly electricity price increase	[%]		3.50	
Yearly gas price increase	[%]	4.50		
Operating hours per operating year	[h]		3'000	
Operating years	[a]		20	
<b>TOTAL CAPEX before operations begin (incl. N+R)</b>	<b>[€]</b>	<b>20'699'357</b>	<b>19'834'143</b>	<b>20'942'429</b>
<b>TOTAL OPEX after utilisation period (incl. N+R)</b>	<b>[€]</b>	<b>64'418'993</b>	<b>67'414'596</b>	<b>65'989'210</b>
<b>TOTAL MAEX after utilisation begin (incl. N+R)</b>	<b>[€]</b>	<b>7'212'130</b>	<b>4'491'682</b>	<b>1'380'913</b>
<b>TOTAL LCCs after utilisation period</b>	<b>[€]</b>	<b>92'330'480</b>	<b>91'740'421</b>	<b>88'312'551</b>

\*) only for informative purposes, do not affect the LCC

# Results on case study



## ■ LCCs are mainly defined through OPEX

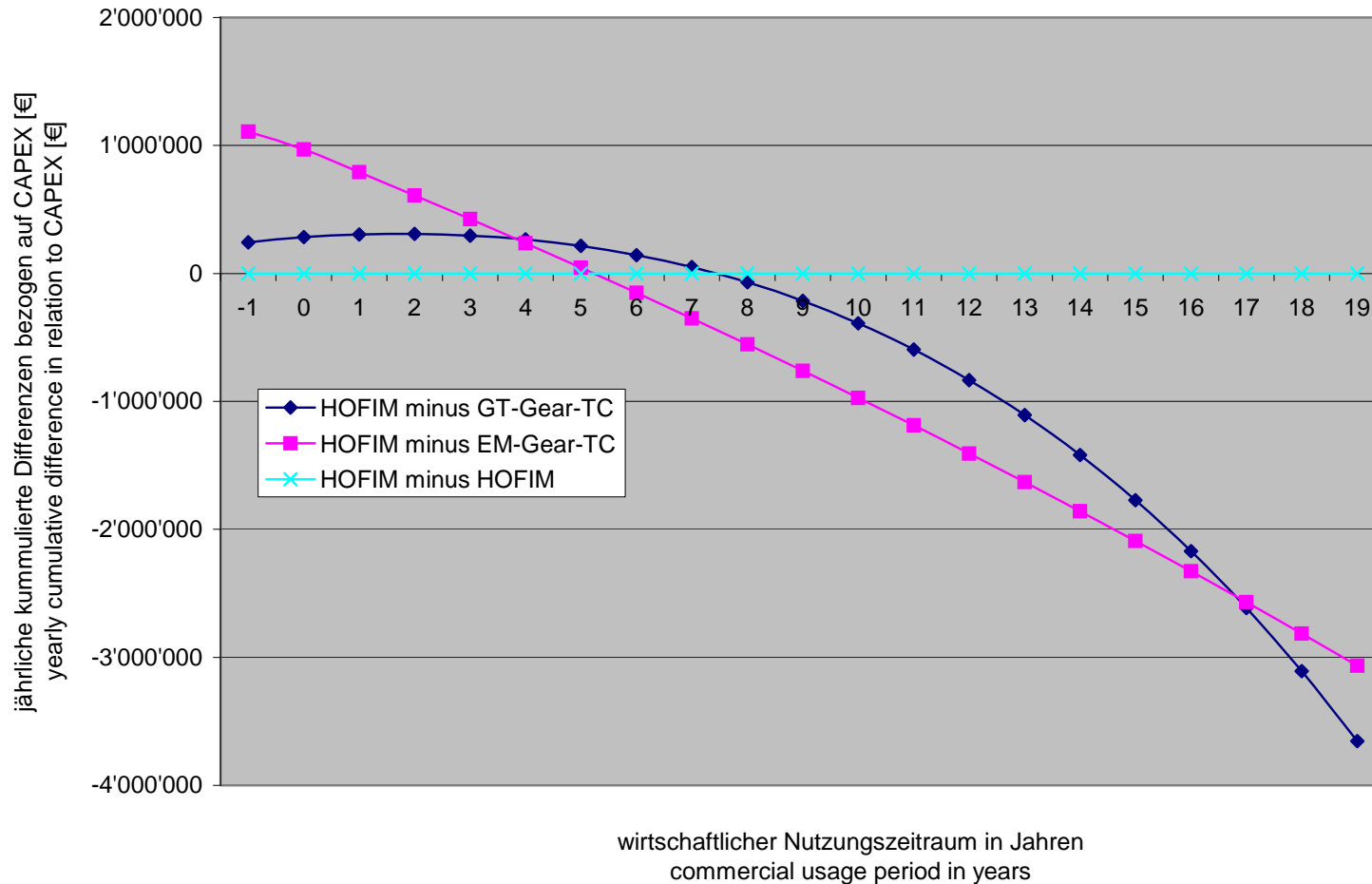


# Case study: Great Britain



## Abgrenzungsanalyse für Gasspeicher Anwendungen Amortisation analysis for a gas storage application

Grossbritannien  
Great Britain



# Expected additional emission costs

## ■ Cost of Emissions

- Beside CO<sub>2</sub>, other greenhouse gases are involved (CH<sub>4</sub>, N<sub>2</sub>, HFCs, SF<sub>6</sub>)
- Greenhouse effect of methane is approx. 23 times CO<sub>2</sub>
- CO<sub>2</sub> – trading already common practice
- Costs of emissions for CH<sub>4</sub> will have to be considered in future (are already considered in several countries)

$$k_{Methan} \approx 23 \times \frac{CO_2}{t} \approx 23 \times Carbix$$

$$K_{Leakage} = \sum_{j=0}^{T-1} m_{K,Leakage} \times t_{Betrieb} \times k_{Methan} \times \left( 1 + \frac{i_{Leakage}}{100} \right)^j$$

# Conclusion



## Electric Compression is of advantage when.....

- a secure and economic source of electric power is available near the compressor station
- capital investment should be as low as possible
- high humidity and ambient temperatures requires derating of the gas turbine
- the electric price/gas price ratio is reasonable
- low-cost night-time energy available for pipeline/storage puffer operation
- environmental restrictions apply (gaseous and noise emission limits, architectural restrictions)
- varying fuel gas qualities or limited transmission capacity in the pipeline exists
- high operating flexibility is required (starting frequency, frequent load cycles, and wide speed range)
- high starting and operating reliability/availability is required
- low operation costs / unmanned operation is required

# Conclusion



A general statement cannot be made, because too many project-dependent variables will influence the result. A detailed LCC cost analysis has to be made for each project individually.

- A decision from CAPEX perspective for fuel or electric driven equipment is generally based on the availability of electric power on site, or the cost to bring the power to site.
- If electric power is available at site, the decision going with fuel/electric driven trains is generally OPEX driven (fuel/electric costs)
- Final decision for electric compression or for gas engine driven compression can only be made after considering all project related costs over the life time of the project.

**FLEXIBILITY and unknown upcoming future EMISSION REGULATIONS are the keywords in today's changing gas storage and gas transport business. Electric Compression is preferred to cope with these uncertainties.**

A close-up, high-angle photograph of several turbocharger compressor wheels. The wheels are made of polished metal and are arranged in a row, showing their complex, curved blades and the central hub. The lighting creates strong highlights and shadows, emphasizing the metallic texture and the precision engineering of the components.

**Thank you for your attention.**

**[matthias.grapow@man.eu](mailto:matthias.grapow@man.eu)**